

11/11 gewerkte  
versie

**Exam Regeltechniek TBKRT05E**

**Tuesday 6 November 2012, 9:00 - 12:00 uur**

Name:

Student ID:

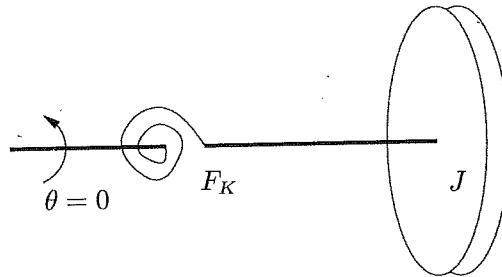
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- This exam consists of 14 pages with 7 open questions. Check if you have all pages.
  - The answers to the questions (including motivation for ALL answers) have to be placed in the answer boxes.
  - Please put your name and student number at all pages. The exercises will be collected separately.
  - If you like, you can add additional paper, which need to include your name and student number. Please provide SEPARATE papers for separate exercises.
  - The use of book, reader and course material is allowed, under the condition that everything is tight into one bundle. E-readers, PDA's, tablets, etc., are NOT allowed.
  - To obtain a grade, the practical has to have been finalized in 2011 or 2012.
  - Good luck!

**Excercise 1 (18 points)**

Given is a rotating mass connected to a rotating spring (i.e., instead of the displacement, the spring works on the angular displacement). The mass has an inertia  $J$  and the rotational spring is nonlinear with spring force

$$F_K = K\theta + \frac{K}{4}\theta^3.$$

The left side of the spring is connected, and thus has angle  $\theta = 0$ . An external torque  $\tau$  is applied to the rotating mass.



- a). Provide the Lagrangian and the Euler-Lagrange equations. Motivate your answer!

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$$L(\theta, \dot{\theta}) = \frac{1}{2} J \dot{\theta}^2 - \frac{1}{2} K \theta^2 - \frac{K}{4} \theta^4$$

$$\frac{\partial R}{\partial \dot{\theta}} = J \dot{\theta} \quad \frac{\partial L}{\partial \dot{\theta}} = J \dot{\theta} \quad \frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} - \frac{\partial L}{\partial \theta} = \tau \Rightarrow$$

$$J \ddot{\theta} + K \theta + \frac{1}{4} K \theta^3 = \tau$$

- b). Now assume that the rotating mass is damped by a linear friction force (linear with the angular velocity) with constant  $b$ . Provide the Rayleigh dissipation function and the corresponding Euler-Lagrange equations. Motivate your answer!

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$$R(\dot{\theta}) = \frac{1}{2} b \dot{\theta}^2 \quad \frac{d}{dt} \frac{\partial L}{\partial \dot{\theta}} - \frac{\partial L}{\partial \theta} = \tau - \frac{\partial R}{\partial \dot{\theta}} \text{ Hence}$$

$$\frac{\partial R}{\partial \dot{\theta}} = b \dot{\theta}$$

$$J \ddot{\theta} + b \dot{\theta} + K \theta + \frac{1}{4} K \theta^3 = \tau$$

- c). Determine a state space model from the Euler-Lagrange equations. Motivate your answer!

For example choose  $x_1 = \theta$ ,  $x_2 = \dot{\theta}$ , then  
and  $u = \tau$

$$\begin{cases} \dot{x}_1 = x_2 \\ \dot{x}_2 = -\frac{b}{J}x_2 - \frac{K}{J}x_1 - \frac{K}{4J}x_1^3 + \frac{1}{J}u \end{cases}$$

↳ follows from the EL equation of b)

- d). Now take as spring force a linear rotational spring, i.e.,  $F_K = K\theta$ . Take  $J = 2$ ,  $K = 3$  en  $b = 5$  and take as input the torque  $\tau$  and as output the angular position of the rotating mass. Determine the transfer function. Motivate your answer!

linear spring force, hence state space model

$$\begin{cases} \dot{x}_1 = x_2 \\ \dot{x}_2 = -\frac{b}{J}x_2 - \frac{K}{J}x_1 + \frac{1}{J}u \end{cases} \quad A = \begin{pmatrix} 0 & 1 \\ -\frac{K}{J} & -\frac{b}{J} \end{pmatrix} \quad B = \begin{pmatrix} 0 \\ \frac{1}{J} \end{pmatrix}$$

$$y = x_1, \quad C = (1 \quad 0) \quad D = 0$$

With  $J=2$ ,  $K=3$ ,  $b=5$   $A = \begin{pmatrix} 0 & 1 \\ -\frac{3}{2} & -\frac{5}{2} \end{pmatrix}$ ,  $B = \begin{pmatrix} 0 \\ \frac{1}{2} \end{pmatrix}$ ,  $C = (1 \quad 0)$

$$(sI - A)^{-1} = \begin{pmatrix} s & -1 \\ \frac{3}{2} & s + \frac{5}{2} \end{pmatrix}^{-1} = \frac{1}{s(s + \frac{5}{2}) + \frac{3}{2}} \begin{pmatrix} s + \frac{5}{2} & 1 \\ -\frac{3}{2} & s \end{pmatrix}$$

$$C(sI - A)^{-1}B = \frac{1}{s + \frac{5}{2}} \begin{pmatrix} s + \frac{5}{2} & 1 \\ -\frac{3}{2} & s \end{pmatrix} \begin{pmatrix} 0 \\ \frac{1}{2} \end{pmatrix} = \frac{1}{2s^2 + 5s + 3}$$

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**Excercise 2 (18 punten)**

Consider a model of the DC motor with neglectable inductance and nonlinear damping (drag). If the inductance is very small:

$$J_m \ddot{\theta}_m + b \dot{\theta}_m^2 = \frac{K_m}{R_a} v_a - \frac{K_m K_e}{R_a} \dot{\theta}_m$$

with motor voltage  $v_a$ , motor speed  $\omega_m = \dot{\theta}_m$ ,  $J_m$  is the inertia,  $K_m$  the motor constant for the mechanical connection with the electrical part,  $K_e$  the motor constant for the electrical connection with the mechanical part,  $R_a$  the electrical resistance, and  $b$  a constant corresponding to the nonlinear damping.

- a). Take as state  $x = \omega_m$ , input  $u = v_a$ , and output  $y = \omega_m$ . Write the state space equations, and determine the equilibrium point(s) for  $u = 1$ . Motivate your answer!

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$$x = \dot{\theta}_m$$

$$\dot{x} = -\frac{b}{J_m} x^2 - \frac{K_m K_e}{J_m R_a} x + \frac{K_m}{J_m R_a} u$$

$$y = x$$

$\dot{x} = 0$  for  $u=1 \Leftrightarrow b x^2 + \frac{K_m K_e}{R_a} x = \frac{K_m}{R_a}$

hence, two eq. points.  $(\Rightarrow) x = -\frac{K_m K_e}{2 R_a b} \pm \frac{1}{2b} \sqrt{\frac{K_m^2 K_e^2}{R_a^2} + \frac{K_m 4b}{R_a}}$

- b). Linearize the system around the point  $u = 0, x = 0$ . Is this linear system stable, asymptotically stable or instable? Motivate your answer!

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$u=0, x=0$ . If  $u=0$ , then  $x=0$  eq. point.

$$f(x) = -\frac{b}{J_m} x^2 - \frac{K_m K_e}{J_m R_a} x$$

$$g(u) = B = \left( \frac{K_m}{J_m R_a} \right)$$

$$h(x) = C = 1$$

$$D = 0$$

$$\frac{\partial f}{\partial x} \Big|_{(0,0)} = \left( -\frac{2b}{J_m} x - \frac{K_m K_e}{J_m R_a} \right) \Big|_{(0)} = -\frac{K_m K_e}{J_m R_a} = A$$

If  $K_m, K_e, J_m, R_a > 0$ , then  $A < 0$ , hence as. stable.

Please turn over

~~$\dot{x} = Ax + Bu$~~   
 ~~$y = Cx$~~

$$\Delta \dot{x} = A \Delta x + B \Delta u$$

$$\Delta y = C \Delta x$$

$$\Delta x = x - x^* = x$$

$$\Delta u = u - u^* = u$$

- c). Linearize the system about the point  $u = 1, x = -1$  and  $u = 1, x = 0.5$ . Are these linear systems stable, asymptotically stable or unstable? Motivate your answer!

The B and C matrices are the same  
(1x1 matrices, or in other words scalars)

$$B = \frac{k_m}{J_m R_a}$$

$$C = 1$$

$$A_1 = \left. \frac{\partial f}{\partial x} \right|_{x=-1} = \frac{2b}{J_m} - \frac{k_m k_e}{J_m R_a}$$

$$\Delta \dot{x}_1 = A_1 \Delta x_1 + B \Delta u$$

$$\Delta y_1 = C \Delta x_1$$

$$\Delta x = x - x^* = x + 1$$

$$\Delta u = u - 1$$

depends on values for

$$A_2 = \left. \frac{\partial f}{\partial x} \right|_{x=0.5} = -\frac{b}{J_m} - \frac{k_m k_e}{J_m R_a} < 0$$

$$\Delta \dot{x}_2 = A_2 \Delta x_2 + B \Delta u \quad \text{if } J_m, k_m, k_e, R_a > 0$$

$$\Delta y_2 = C \Delta x_2 \quad \text{and hence as. stable}$$

$$\Delta x_2 = x - 0.5 \quad \Delta u = u - 1$$

if  $\frac{k_m k_e}{R_a} > 2b$  for  $J_m, k_m, k_e, R_a > 0$  then as. stable.

if  $\frac{k_m k_e}{R_a} = 2b$  then stable

if  $\frac{k_m k_e}{R_a} < 2b$  then unstable.

**Excercise 3 (18 punten)**

Consider the following continuous time state space system with input  $u(t)$ , and a constant  $a$ .

$$\dot{x}(t) = \begin{pmatrix} 1 & 5 \\ a & -5 \end{pmatrix} x(t) + \begin{pmatrix} 1 \\ 0 \end{pmatrix} u(t)$$

a). For which  $a$  is this system stable, asymptotically stable, unstable? Motivate your answer!

$$\det(\lambda I - A) = (\lambda - 1)(\lambda + 5) - 5a = 0$$

$$\Leftrightarrow \lambda^2 + 4\lambda - 5 - 5a = 0$$

$$\Leftrightarrow \lambda = -2 \pm \frac{1}{2} \sqrt{16 + 20 + 20a}$$

$$\operatorname{Re}\{\lambda\} < 0 \quad \text{if} \quad 20 + 20a < 0 \quad \Leftrightarrow \quad a < -1 \rightarrow \text{as stable}$$

$$\operatorname{Re}\{\lambda\} = 0 \quad \text{if} \quad a = -1 \quad \rightarrow \text{stable}$$

$$\operatorname{Re}\{\lambda\} > 0 \quad \text{if} \quad a > -1 \quad \rightarrow \text{unstable}$$

b). Determine for which  $a$  the system is reachable. Motivate your answer!

$$W_r = [B \quad AB] = \begin{bmatrix} 1 & 1 \\ 0 & a \end{bmatrix}$$

4  $\det W_r \neq 0$  if  $a \neq 0$ . Hence  
reachable if  $a \neq 0$ , not reachable  
if  $a = 0$

Please turn over

- c). Determine the state feedback  $u = -Kx + k_r r$  for  $a = 1$  such that the closed loop system is stable for reference signal  $r$ , and such that the closed-loop poles are given by  $-1$  and  $-2$ . Motivate your answer!

You can follow Theorem 6.3 from the book:

$$K = [p_1 - a_1 \quad p_2 - a_2] \tilde{W}_r W_r^{-1}$$

Char. polynomial of  $A$  matrix:  $\lambda^2 + u\lambda - 10$

$$\Rightarrow a_1 = 4, \quad a_2 = -10$$

Char. polynomial of closed loop:  $(\lambda + 2)(\lambda + 1) = \lambda^2 + 3\lambda + 2$

$$\Rightarrow p_1 = 3, \quad p_2 = 2$$

$$\tilde{W}_r = \begin{pmatrix} 1 & 4 \\ 0 & 1 \end{pmatrix}^{-1} = \begin{pmatrix} 1 & -4 \\ 0 & 1 \end{pmatrix}$$

$$W_r \text{ from exercise b): } W_r = \begin{pmatrix} 1 & 1 \\ 0 & 1 \end{pmatrix}$$

$$\Rightarrow K = [-1 \quad 12] \begin{pmatrix} 1 & -4 \\ 0 & 1 \end{pmatrix} \begin{pmatrix} 1 & -1 \\ 0 & 1 \end{pmatrix} = [-1 \quad 12] \begin{pmatrix} 1 & -5 \\ 0 & 1 \end{pmatrix} = [-1 \quad 17]$$

In addition  $k_r = -1/C(A-BK)^{-1}B =$  with output  $x_1$   
 $C = [1 \quad 0]$

$$-1 / [1 \quad 0] \begin{pmatrix} 2 & -12 \\ 1 & -5 \end{pmatrix}^{-1} \begin{pmatrix} 1 \\ 0 \end{pmatrix} = -\frac{1}{[1 \quad 0] \begin{pmatrix} -2\frac{1}{2} & 6 \\ -\frac{1}{2} & 1 \end{pmatrix} \begin{pmatrix} 1 \\ 0 \end{pmatrix}} = -\frac{1}{2\frac{1}{2}} = -\frac{2}{5}$$

- d). Is the closed loop system underdamped, overdamped or critically damped? Motivate your answer!

Char. polynomial  $\lambda^2 + 3\lambda + 2$

$$\Leftrightarrow \omega_n^2 = 2 \Leftrightarrow \omega_n = \sqrt{2} \quad 2\zeta\omega_n = 3 \Leftrightarrow \zeta = \frac{3}{2\sqrt{2}}$$

closed loop

$$\approx 1,067$$

hence the system is overdamped!



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**Exercise 4 (8 points)**

Given a transfer function of the closed loop system as

$$T(s) = \frac{b_0 s + b_1}{s^2 + a_1 s + a_2}$$

The controller was designed such that the closed loop system has a DC gain (the gain for  $\omega = 0$ ) equal to 1 and a rise time  $t_r < 0.2 \text{ sec}$  (where the rise time  $t_r$  is assumed to be equal to the rise time of the system  $\frac{1}{s^2 + a_1 s + a_2}$ ). What can you conclude for the system parameters  $b_0, b_1, a_1, a_2$  so that  $T$  fulfills these specifications? Motivate your answer! (hint: not all parameters are restricted by these specifications).

Recall that  $t_r \approx \frac{1.8}{\omega_n}$        $\omega_n = \sqrt{a_2}$

Hence  $t_r = \frac{1.8}{\sqrt{a_2}} < 0.2 \Rightarrow a_2 > \frac{81}{4}$  4 punten

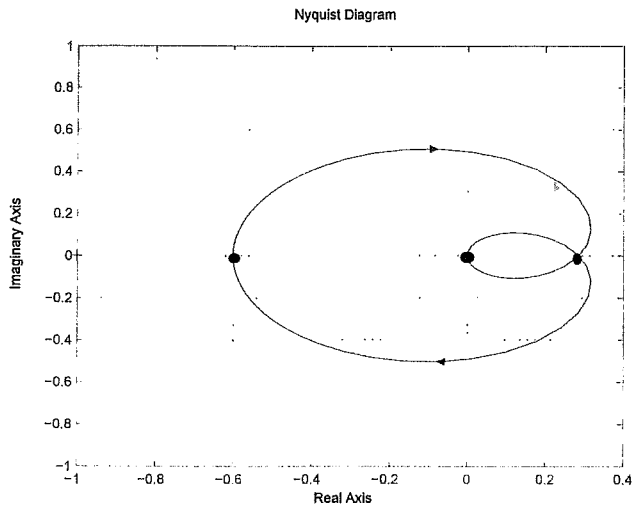
DC gain  $T(0) = \frac{b_1}{a_2} = 1 \Rightarrow b_1 = a_2$

hence  $b_1 > \frac{81}{4}$  4 punten.

Please turn over

**Exercise 5 (15 points)**

Consider the Nyquist plot of a transfer function  $G(s)$  of a stable system as follows:



- a). For which values of  $K$  is the standard closed loop system for loop transfer  $L(s) = KG(s)$  stable? Motivate your answer!

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$GM = \frac{1}{0.6}$ . Hence for  $k > \frac{1}{0.6} \approx 1.7$  the system becomes unstable since it then will encircle  $-1$ . Stable for  $k < 1.7$ .

- b). Assume that  $K = 5$ . Provide  $N$ ,  $P$  and  $Z$  from the Nyquist criterium. Is the closed loop system stable? Motivate your answer!

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See a),  $k = 5 > 1.7$ , hence unstable

$G$  is stable, hence  $P = 0$

Number of encirclements of  $-1$ :  $N = 1$

Hence  $Z = N + P = 1$ , the closed loop system has 1 unstable pole.

N=2pt

P 2

Z 1

correct

Z=0 unstable 3pt

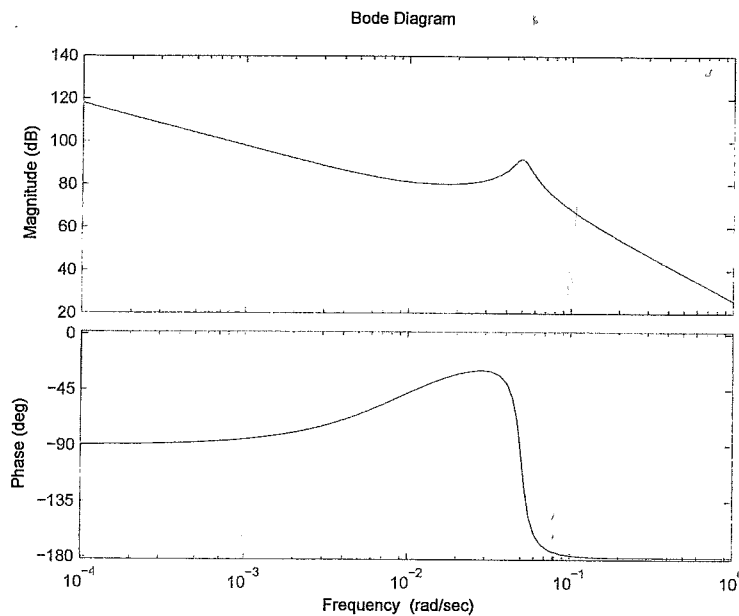
unstable pole 1pt

**Exercise 6 (15 points)**

For the design of a controller for an airplane the following model is being used:

$$G(s) = \frac{80(100s + 1)}{s(400s^2 + 4s + 1)}$$

The Bode plot of the system is given by



The system can be treated as a second order system. Design a lead controller

$$D(s) = K \frac{Ts + 1}{\alpha Ts + 1}$$

such that

- The cross-over frequency  $\omega_c = 0.1$ .
- The phase margin  $PM \geq 50$ .

Hint: first determine  $\alpha$ , then  $T$  and then  $K$ . Motivate your answer!

A  $\omega_c = 0.1$ ,  $50^\circ$  <sup>phase lead</sup> needs to be added. Relation between phase lead and  $\alpha$  is given

$$\text{Hence } \alpha = \frac{1 - \sin 50^\circ}{1 + \sin 50^\circ} \approx \frac{1}{8} \approx 0.13$$

Please turn over

1,71 voor 50 rad

Furthermore, for a lead we have maximum phase addition at

$$\omega = \frac{1}{T\sqrt{\alpha}}$$

which we want to be the new cross over frequency. Hence

$$\omega_c = 0.1 = \frac{1}{T\sqrt{\alpha}} \Rightarrow T \approx 20,3$$

Now the appropriate cross over frequency is determined by adjusting  $K$ , i.e.,

for  $\omega = 0.1$ , we have  $|G(0.1j)| = 2,663 \times 10^3$   
(either calculated or from plot)

and  $|D(0.1j)| = K \cdot 2,663$ . To have  $\omega_c = 0.1$ , it needs to be that

$$|G(0.1j)| |D(0.1j)| = 1 \Leftrightarrow K \approx 1,33 \cdot 10^{-4}$$

	valledig	geen	motivate	alleen correcte formule
$\alpha$	spt	3		1
$T$	spt	3		1
$K$	spt	3		1

**Exercise 7 (12 points of which 4 are bonus)**

Consider for example the simple model of a DC motor given by

$$G(s) = \frac{1}{s(s+1)}$$

Explain when you would advice to design a lead compensator, and when to design a lag compensator. Provide a short and simple manual for engineers to help them decide what to design, and based upon which principles.

See slides lecture 12 of book. For example:

\* A lead network increases the ratio  $\frac{\omega_c}{DG(0)}$   
 hence if  $\omega_c$  is constant the low frequency gain is decreasing, or if the low frequency gain is constant,  $\omega_c$  increases

\* A lag network decreases the ratio  $\frac{\omega_c}{DG(0)}$ ,

hence precisely opposite to the above.

In general,  $\omega_c$  large is not so good. (amplification of high frequency noise!)

~~hence~~ This is linked to

bandwidth and other consideration

(often the higher frequencies correspond to noise and disturbances, and then we want the magnitude to be small).

END EXAM